

Design Study for Single Stage High Pressure Turbine of Gas Turbine Engines

Ajoko, Tolumoye John

*Department of Mechanical/Marine Engineering, Faculty of Engineering,
Niger Delta University, Wilberforce Island, Bayelsa State, Nigeria.*

Abstract: - The research paper is a design study to reduce multiple stages of High Pressure Turbine (HPT) to maintain the same thrust – to – weight ratio of gas turbines. This current approach of gas turbine design is to reduce cost and weight of the component. The preliminary design for the turbomachinery features three different gas turbines such as AL-2LF-3, GT – 26, and SK30 – GT. The research survey used to fulfill this task is an Advance Mathematical Modeling Principles; based on the Inlet Annulus Design Analysis, Prediction of Turbine Efficiency using Smith's Efficiency Correlation Chart, Design Analysis for the Outlet Annulus and a design study for Turbine Free Vortex. The ability to determine this aerodynamic geometry of the HPT stage(s) of gas turbine is the peak of the research.

Nevertheless, study results revealed that a single stage HPT deriving a corresponding compressor can produce the same and needed aerodynamic performance of the gas turbine. Thus, all conditions required in the design of HPT stage were met having turbine stage efficiency within the range of $1.0 < (\Delta H/U^2) < 2.5$ and $0.5 < (Va/U) < 0.8$; fulfilling Smith's Efficiency Prediction Law. A corresponding Mach number for the three engines of study are 0.51, 0.46 and 0.52 respectively. This is a clear indication to prevent the choking condition of the compressible flow at the minimum area along the duck of the gas turbine.

Keywords: - HPT, Mach Number, stage isentropic efficiency, Thrust(Power), Turbomachinery

I. INTRODUCTION

Developing and acquiring good knowledge of the functions of an axial high pressure turbine (HPT) component is important to its designers and proficient users that are accessible to the performance of gas turbine engines. The ambitious performance role of any axial turbine component of gas turbine engine is to drive a corresponding axial compressor for gas pipeline or external load purposes. Thus, this understanding will expose designers the advantages and limitations likely to be encounter in its design/manufacturing processes. One major objective of these preliminary aerodynamic design studies of the axial turbine is targeted to yield high isentropic efficiency for the turbine, because the technical quality of any machine is best described by its efficiency [1]. Another challenging constrains leading to this study is the weight, cost, fuel consumption, emissions from engine, durability and how reliable the gas turbine engine is, regardless of its area of application.

Considerable solution to remedy these limitations is the reduction of turbine stages to produce the same output thrust or power for aero and industrial gas turbine machines respectively. To maintain availability of the designed axial high pressure turbine component of gas turbine, the test and comparison of specific fuel consumption to the initially manufactured engine is necessary. Many scholars have contributed excellently on the design analysis for gas turbine components. According to [2] preliminary detail calculation of gas turbine components helps the designer to understand the characteristic dimensions and gas angles in compressor stages using a real gas turbine model. This analysis allows the estimation of inlet and outlet cross-sectional area and number of the corresponding stages needed in a compressor. However, designing a turbine component to attain high thermodynamic efficiency, heat addition into the inlet temperature of the component should be as high as possible [3]. This employs a corresponding cooling component to cater for any excess or over-heating problems. Meanwhile, there is an inherent exchange between increasing a cycle temperature and the cooling penalties in the cycle; knowing that cooling flows can impact the overall thermal efficiency of the engine, therefore losses that increase cooling flow can have a cycle penalty [3].

The current approach in aero gas-turbine engine design is to increase the thrust-to-weight ratio and stage pressure ratio leading to compressor design with higher aerodynamic loads and reducing the number of blades and stages and thus diminishing the overall size and weight of the machine. Hence, the pressure rise per stage and the efficiency must be increased and aerodynamic stability of a compressor is limited by the behavior of the tip leakage flows or the hub corner stall when the operating point gets closer to the stall or surge limit [4,5]. Knowing the performance role of a turbine component in gas turbine engine as a compressor driving mechanism, may also needs a corresponding design to meet the target of its counterpart. The advancement of turbo-compression technology is a reflection of a higher work capacity per stage as a result of increases in rotor speed, aerodynamic loading; meanwhile an incremental performance enhancement can be made through geometric optimization and improved design methods [6,7].

The optimization techniques give direct control on the performance parameters of the gas turbine which allows the designer to explore the design space to achieve a given objective. One possible design objective is to minimize flow losses, which can be measured by the total pressure loss (or entropy generation) [8]. Thus, minimizing flow losses can be achieved by proper reshaping of the blade profile. The consistency in gas turbine operation relies on the structural integrity of its rotating parts. Thus, the design analysis and testing of component cycle is intensely a rigorous phase that ensures the well-being of the final product satisfying the requirement of the manufacturer [9]. This design process of an axial flow turbine still remains a very complex, fussy, multidisciplinary task where aero-thermodynamic issues, aero-mechanical, technological, structural, noise related cases, emission and other prevalent matters are considered simultaneously which leads to very challenging problems for the designer. This is true fact mostly for aircraft engines with stringent demands for low weight, high strength and extended life [10,11].

In respect of handling this complexity, the stage designing process of an axial high pressure turbine component of gas turbine engine as treated in this paper is supported with an Advance Mathematical Modeling Principles and conclusion is drawn based on Smith's Efficiency Correlation method of turbine stage efficiency prediction.

II. GEOMETRIC DESCRIPTION OF ENGINES OF STUDY

The general arrangement for turbine component representing a HPT used for model development is shown in figure 1 below. Both single and double spool aero gas turbines are considered for the purpose of this study. The importance of this research is to carry out a feasibility test for a single stage preliminary design for the HPT.

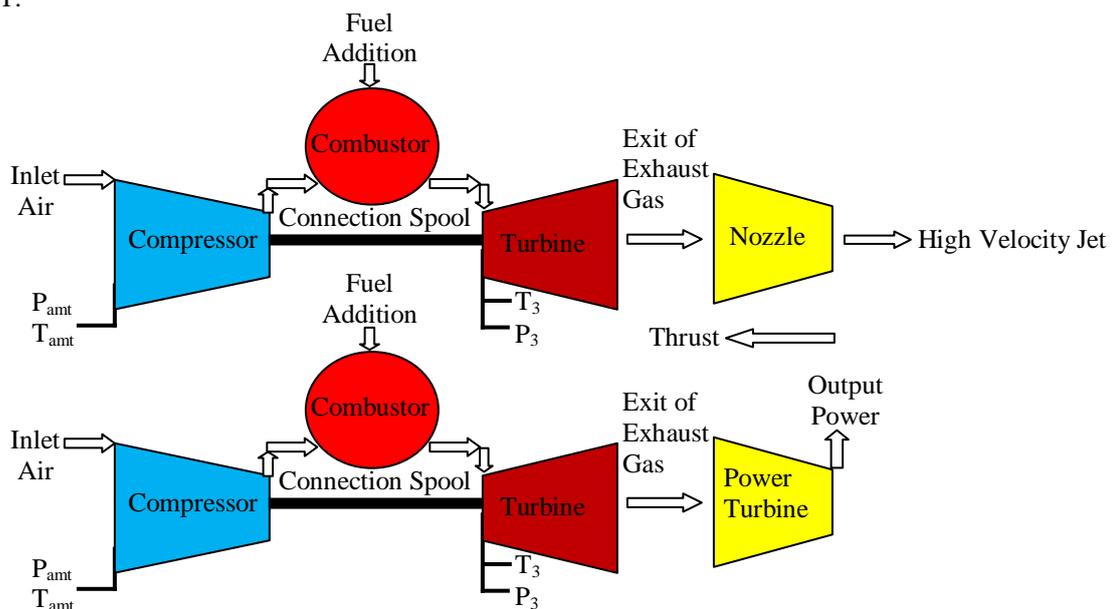


Figure 1: A simple Gas Turbine Cycle as Aero and Industrial one Spool Gas Turbine Engines

III. DESIGN SPECIFICATIONS

The design specifications for the engines of study are in table 1 which are available in [12,13,14] highlights the needed parameters for subsequent design analysis. However, in the design process of the axial turbine, some assumptions were considered. They are such as:- Constant Vortex Flow, 50% Reaction at Blade Mid Height, Straight Sided Annulus Walls, Constant Axial Velocity, Constant Mean Diameter and a Constant Shaft Speed.

Table 1: Turbine Design Specifications

Manufacturer	Model	Turbine Inlet Flow Mach No	Mean Diameter (Dm) m	Shaft Speed (rpm)	Polytropic Efficiency	TET (K)	Turbine Inlet Mass Flow(kg/s)	Turbine Work (TW) (MW)	Specific Heat (Cp) KJ/KgK	Gamma
LYULKA	AL-2LF-3	0.3	1.72	3000	0.9	562	25.497	3.84	1277	1.29
ALSTOM	GT-26	0.3	2.95	3000	0.9	1757	622.79	288.3	9756	1.4
ROLLS-ROYCE	SK30 - GT	0.3	2.75	3001	0.9	1158.34	93.6	20	3562	1.32

IV. DESIGN ANALYSIS

The designing of HP turbine with high engine thermal efficiency is usually constrained by high temperature environment, thereby needs a subsequent cooling effect. Also, the load carrying capacity of the turbine disc is a major concern. Therefore, designing this component needs some harmonizing technology to put all these limitations into consideration for suitable and effective design work. With respect to this study, some design parameters were calculated from the performance specifications of the real engine as classified on table 1 above.

They are the hot mass flow from the combustor to the inlet of the turbine, the turbine entry temperature (TET), turbine work (TW) etc; whereas a preliminary assumption of the engine ambient condition and fuel caloric value (FCV) are taken as 101.325KPa, 288K and 43MJ/Kg, respectively. Meanwhile, a 5% combustor pressure loss and 2% pressure drop at turbine exit are considered likewise.

V. INLET ANNULUS GEOMETRY

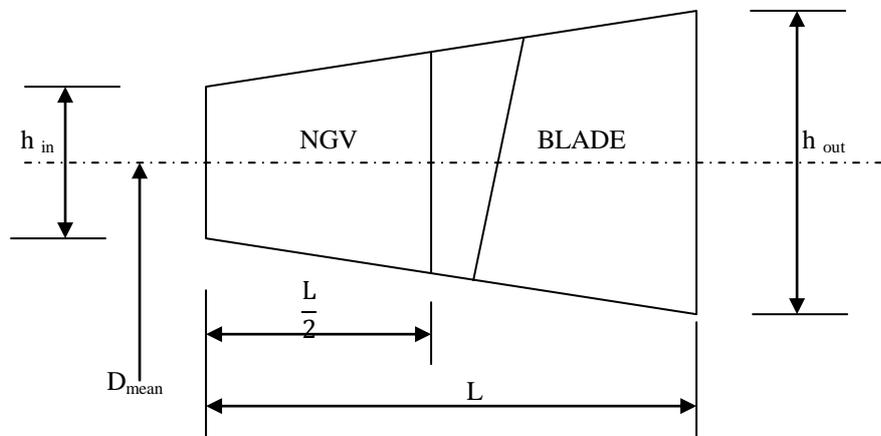


Figure 2: High Pressure Turbine Annulus Diagram

The governing equations with respect to the calculation of the Inlet Annulus Geometry of an axial HP turbine is given in equations (i) to (v):

$$P_3 = PR \cdot P_{amt} \cdot \text{Percentage of combustor pressure loss} \dots\dots\dots (i)$$

$$A = \frac{W \cdot \sqrt{T_3}}{Q \cdot P_3} \dots\dots\dots (ii)$$

$$h = \frac{A}{\pi \cdot D_m} \dots\dots\dots (iii)$$

$$D_{tip} = D_m + h \dots\dots\dots (iv)$$

$$D_{hub} = D_m - h \dots\dots\dots (v)$$

VI. ANALYSIS FOR THE PREDICTION OF TURBINE EFFICIENCY

The isentropic efficiency of the component is generally used as a true measure of the engine’s performance. Hence, Smith’s chart correlation method of turbine efficiency prediction is considered for the study. This industrial based technique recognized that whilst the designer would always attempt to minimize loss components, the major factors affecting turbine efficiency would always be design levels of gas velocity and deflection.

According to the Smith’s correlated turbine efficiency chart, measurements from stage loading coefficient ($\Delta H/U^2$) and flow coefficient (V_a/U) are connected and traced to the efficiency correlation curves; where the best turbine efficiency are taken. Meanwhile, Smith’s Chart correlation always recommends a value of minus 2% from the read value for a test of accuracy and best stage design efficient for turbine. Equations (vi) to (viii) are analytical expressions used for the determination of axial turbine efficiency:

$$\Delta T = \frac{\text{Turbine Power}}{W \cdot C_p} \dots\dots\dots(vi)$$

$$U_{\text{mean}} = \frac{\text{RPM} \cdot \pi \cdot D_m}{60} \dots\dots\dots(vii)$$

$$\Delta H/U^2 = C_{p\text{gas}} \cdot \Delta T / U^2 \dots\dots\dots(viii)$$

VII. OUTLET ANNULUS GEOMETRY

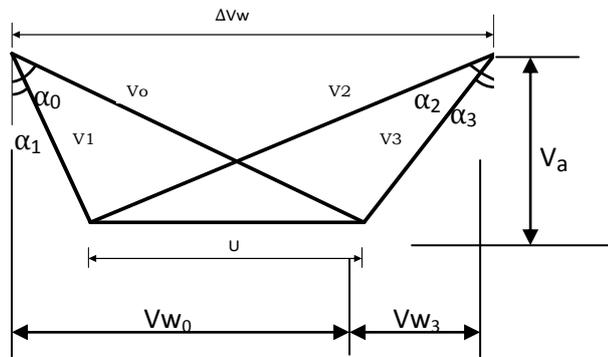


Figure 3: Velocity Diagram

$$\Delta V_w = \left[\frac{\Delta H}{U^2} \right] \cdot \frac{U}{\Omega} \dots\dots\dots(ix)$$

$$V_{w3} = \Delta V_w - U_{\text{mean}}/2 \dots\dots\dots(x)$$

$$\alpha_3 = \tan^{-1} (V_{w3}/V_a) \dots\dots\dots(xi)$$

$$v_3 = V_a / \cos \alpha_3 \dots\dots\dots(xii)$$

$$R = \left[1 - \frac{\Delta T}{\eta_{\text{isent}} \cdot x \cdot T_{\text{in}}} \right]^{Y/(Y-1)} \dots\dots\dots(xiii)$$

$$P_3 = P_{\text{in}} \times R \dots\dots\dots(xiv)$$

$$A_{\text{ann}} = A_3 / \cos \alpha_3 \dots\dots\dots(xv)$$

In order to increase a design power, there will be increase in NGV turning and exit whirl angle. However, the process stage require cooling for the production of high thermal efficiency will keep the trailing edge of the high (α_0) NGV to be thinner and this occurrence happens when $\alpha_0 = 70 - 72^\circ$ approximately [11]. The above mathematical expressions, equations (ix) – (xv)] are the basic will-power for outlet annulus geometry analysis.

VIII. RESULTS

Results are presented in the tables (tables 2 – 5) using the relationships provided from the governing equations above. Meanwhile, appropriate values of the mass flow function Q or ($W\sqrt{T}/AP$), velocity function Q ($V_a / \sqrt{T_3}$) and stage isentropic efficiency (η_{isent}) of turbine component are extracts from isentropic flow chart of dry air and Smith’s Efficiency correlation chart [15]. These charts are equally presented in Annex – A; whereas, a work done factor (Ω) of 0.98 was considered in the analysis.

Table 2: Inlet Annulus Design

Engine Model	Pressure Ratio	Ambient Pressure (P ₁)	5% Pressure loss @ CC	Turbine Inlet Pressure (P _{in})	Values of Q For M = 0.3	Turbine Inlet Mass Flow	TET	Annulus Area A	Annulus Mean Diameter Dm	Height of Annulus h	Tip Diameter Dtip	Hub Diameter Dhub	Dhub/Dtip Ratio
LYULKA AL-21F-3	3.6	101325	0.95	346532	0.019	25.5	562	0.09	1.72	0.017	1.74	1.71	0.981
ALSTOM GT - 26	33.9	101325	0.95	3263172	0.020	622.8	1757	0.41	2.95	0.044	2.99	2.91	0.971
Rolls Royce SK30 GT	11.0	101325	0.95	1058846	0.019	93.6	1158	0.16	2.75	0.018	2.77	2.73	0.987

Table 3: Prediction of Turbine Efficiency

Engine Model	power (MW)	Mass flow (kg/s)	Specific Heat CP; (kJ/kgK)	Temp Drop ΔT; (°K)	Shaft Speed (rpm)	Dmean (m)	Umean (m/s)	ΔH/U ²	Va/√T ₃ from table	TET	Va	Va/Umean	ηisent (%) (from Smith's Chart)
LYULKA AL-21F-3	3.84	25.5	1277	117.94	3000	1.72	270.21	2.062	5.74	562	135.96	0.503	89.0
ALSTOM GT - 26	288.3	622.8	9756	47.45	3000	2.95	463.45	2.155	5.96	1757	249.82	0.539	88.5
Rolls Royce SK30 GT	40	93.6	1158.34	368.93	3000	2.75	432.03	2.290	5.80	1545.3	227.92	0.528	87.9

Table 4: Design Analysis for Outlet Annulus

Engine Model	Temp Drop (ΔT)	T ₃	ΔVw	Vw ₃	α ₃	V ₃	V ₃ /√T ₃	M ₃ = value matching to V ₃ /√T ₃ From table	Q ₃ = value matching to V ₃ /√T ₃ From table	P ₃	A ₃	Area of Annulus (A _{ann})	Annulus height h	Drip	Dhub	hub/tip ratio
LYULKA AL-21F-3	117.90	444	568.74	149.26	47.67	201.90	9.58	0.51	0.029	105093.85	0.174	0.259	0.048	1.77	1.67	0.946
ALSTOM GT - 26	47.45	1710	1019.24	277.90	48.05	373.76	9.04	0.46	0.028	2927745.95	0.310	0.464	0.050	3	2.9	0.967
Rolls Royce SK30 GT	368.90	1176	1009.37	288.67	55.64	403.83	11.77	0.62	0.034	286984.61	0.332	0.589	0.068	2.82	2.68	0.952

Table 5: Turbine Free Vortex Design

Engine Model	ROOT	BMH	TIP
LYULKA - AL-21F-3			
D (NGV Exit)	1.69	1.72	1.75
D (ROOT Exit)	1.67	1.72	1.77
Va	135.96	135.96	135.96
Vw ₃ mean	149.26	149.26	149.26
Vw ₀ Mean	419.48	419.48	419.48
Vw ₀ = Vw ₀ Mean • (Dmean /D) D @ NGV Exit	427.53	419.63	411.72
α ₀ = tan ⁻¹ (Vw ₀ /Va)	72.36	72.05	71.73
Vw ₃ = Vw ₃ Mean • (Dmean /D) D @ Rotor Exit	153.54	149.38	145.22
α ₃ = tan ⁻¹ (Vw ₃ /Va)	48.48	47.69	46.89
U = Umean • (D /Dmean); D @ Rotor Exit	262.23	269.74	277.25
V ₀ = Va /cosα ₀	563.91	432.25	300.59
Nozzle Acceleration, Va / V _{in} = V ₀ / Va	4.15	3.18	2.21
V ₁ = √[Va ² + (Vw ₀ - U) ²]	214.03	202.63	191.22
α ₁ =cos ⁻¹ (Va / V ₁)	50.56	47.62	44.68
V ₂ = √[Va ² + (U + Vw ₃) ²]	437.43	440.62	443.81
α ₂ =cos ⁻¹ (Va / V ₂)	71.89	72.03	72.16
Rotor Acceleration, V ₂ / V ₁	2.04	2.18	2.32
ALSTOM - GT - 26			
D (NGV Exit)	2.90	2.95	2.99
D (ROOT Exit)	2.89	2.95	3.01
Va	249.82	249.82	249.82
Vw ₃ mean	277.90	277.90	277.90
Vw ₀ Mean	741.34	741.34	741.34
Vw ₀ = Vw ₀ Mean • (Dmean /D) D @ NGV Exit	753.31	741.53	729.75
α ₀ = tan ⁻¹ (Vw ₀ /Va)	71.65	71.38	71.10
Vw ₃ = Vw ₃ Mean • (Dmean /D) D @ Rotor Exit	282.72	277.98	273.25
α ₃ = tan ⁻¹ (Vw ₃ /Va)	48.53	48.05	47.56
U = Umean • (D /Dmean); D @ Rotor Exit	265.15	269.74	274.34
V ₀ = Va /cosα ₀	1036.18	794.26	552.34
Nozzle Acceleration, Va / V _{in} = V ₀ / Va	4.15	3.18	2.22
V ₁ = √[Va ² + (Vw ₀ - U) ²]	548.38	533.91	519.43
α ₁ =cos ⁻¹ (Va / V ₁)	62.90	62.08	61.25
V ₂ = √[Va ² + (U + Vw ₃) ²]	602.13	602.01	601.88
α ₂ =cos ⁻¹ (Va / V ₂)	65.49	65.49	65.48
Rotor Acceleration, V ₂ / V ₁	1.10	1.13	1.16

Engine Model	ROOT	BMH	TIP
Rolls Royce - SK30 GT			
D (NGV Exit)	2.71	2.75	2.79
D (ROOT Exit)	2.66	2.75	2.84
V _a	227.92	227.92	227.92
Vw ₃ mean	288.67	288.67	288.67
Vw ₀ Mean	720.70	720.70	720.70
Vw ₀ = Vw ₀ Mean • (Dmean / D) D @ NGV Exit	732.17	720.88	709.58
$\alpha_0 = \tan^{-1}(Vw_0/Va)$	72.71	72.45	72.19
Vw ₃ = Vw ₃ Mean • (Dmean / D) D @ Rotor Exit	298.74	289.00	279.26
$\alpha_3 = \tan^{-1}(Vw_3/Va)$	56.55	55.65	54.75
U = Umean • (D / Dmean); D @ Rotor Exit	260.65	269.74	278.83
V ₀ = V _a / cos α_0	945.35	724.64	503.92
Nozzle Acceleration, V _a / V _m = V ₀ / V _a	4.15	3.18	2.21
V ₁ = $\sqrt{[V_a^2 + (Vw_0 - U)^2]}$	514.23	492.60	470.97
$\alpha_1 = \cos^{-1}(Va/V_1)$	67.43	66.33	65.23
V ₂ = $\sqrt{[V_a^2 + (U + Vw_3)^2]}$	593.17	592.56	591.95
$\alpha_2 = \cos^{-1}(Va / V_2)$	67.40	67.375	67.35
Rotor Acceleration, V ₂ / V ₁	1.15	1.20	1.26

IX. DISCUSSION OF RESULTS

The preliminary design results for the axial HPT component performance of the turbomachinery as shown above conforms yielding the best stage efficiency performance from the Smith's chart where the stage loading coefficient ($\Delta H/U^2$) is plotted against the flow coefficient (V_a/U). As seen on the tabulated results and in Annex – A; the stage isentropic efficiency for the three engine of study; AL-2LF-3, GT – 26 and SK30 – GT are 89.0%, 88.5% and 87.9% respectively. Also, this moderate values implies low gas velocities and reduction of excessive frictional losses. This satisfies a single stage arrangement in the HPT component design. It is an indication that multiple stage can be replaced with a single stage arrangement of axial HPT using the ideas of the above systematic design study.

Another point of concern is the corresponding Mach numbers for the three engine of study; that is $M_3 = 0.51, 0.46$ and 0.52 for AL-2LF-3, GT – 26 and SK30 – GT from the outlet annulus design which is less than 1. This shows that the inlet mass flow of the HPT and TET is in order. Thus, acknowledging that one key phenomenon for compressible flow is choking; where a Mach number of 1 is reached at the minimum area along a duct. Again, is the modality of increasing the design power of the component which has to do with the exit whirl angle and the effect of turns in the NGV. In order to maintain these conditions to keep high thermal efficiency production; the gas angle, α_0 design analyses were ensured to be at an approved range of $(70 - 72^\circ)$ according to literature review [11]. Therefore, an estimated result of α_0 from table 5 above for AL-2LF-3 is 72.71° at Root, 72.45° at Blade Mid Height (BMH) and 72.19° at tip; while for GT – 26 is 71.65° at Root, 71.38° at BMH and 71.1° at tip and for SK30 – GT is 72.71° at Root, 72.45° at BMH and 72.19° at tip.

X. CONCLUSION

The line of best efficiency for turbine design indicates that the optimum turbines are at the ranges of: $1.0 < (\Delta H/U^2) < 2.5$ and $0.5 < (V_a/U) < 0.8$; and the maximum range efficiency line follows the Smith's efficiency prediction law: $(\Delta H/U^2) = 6.5(V_a/U) - 2.90$. Results of the HP turbine stage efficiency from study responds to Smith's efficiency figures quoted in his correlation. The HP turbine stages frequently appear on the left hand side of the Smith's chart of the best efficiency line; consequently, it is expected that turbine efficiency appears on same side and this is accomplished from the research study. The Mach number at the blade inlet hub from the design analysis of the study is less than 0.7 as stated in [16] is to ensure that there is acceleration relative to the blade all the way through the blade passage.

Results of NGV exit angle is satisfied because it maintains the range angle of $65^\circ - 73^\circ$ to guarantees decrease in pressure losses. Meanwhile, the analysis of the hub – tip ratio is greater than 0.5 from its design to minimize secondary losses; however, less than 0.85 though with reduced blade height. Therefore, the redesigning of a multiple stages to a single stage HPT component achieves the same performance as the original engine.

XI. ACKNOWLEDGEMENT

I sincerely appreciate all staff and colleagues of the department of Mechanical Engineering and the entire faculty of Engineering in my university, Niger Delta University for their encouragement.

REFERENCES

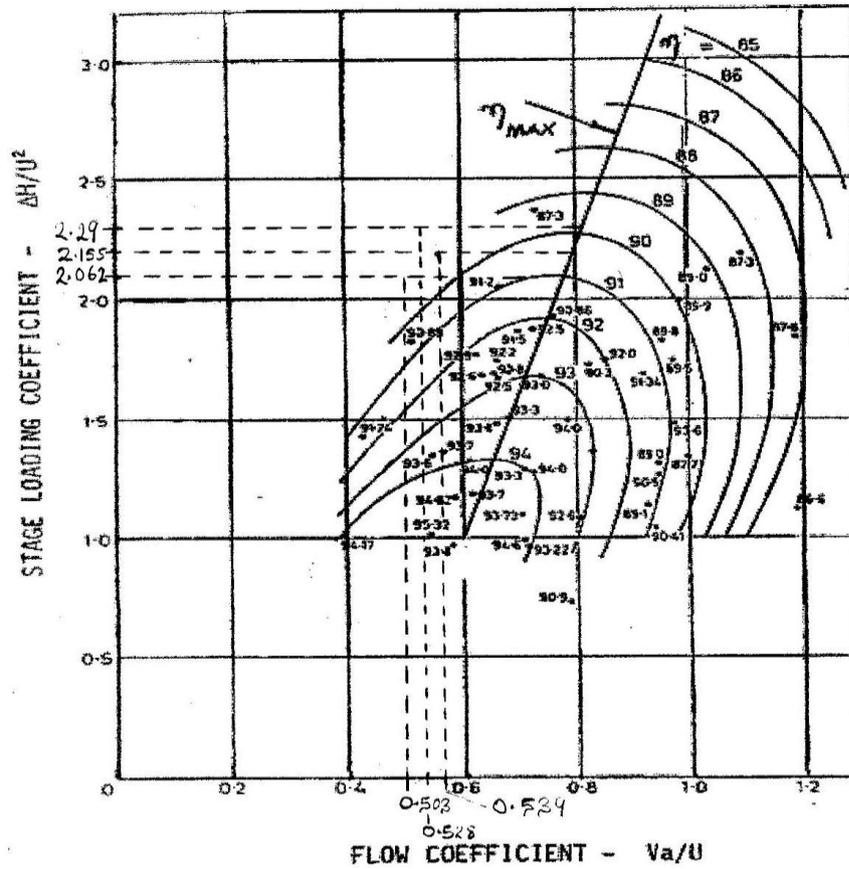
- [1] Neumayer, F., Performance testing of a first and second Transonic Turbine Stages, *ASME TURBO*, 2002, GT-2002-30336.
- [2] Tadeusz, C., Sabastian, L., Sabastain, R., Algorithm for Design Calculation of Axial flow Gas Turbine Compressor – Compression with GTD – 350 Compressor Design, *Institute of Power Engineering and Turbomachinery*, 2011.
- [3] Cyrus, B. M., Mustapher, A. C., Hatim, M. M., Gas Turbine Performance Deterioration, *Proceedings of the 30th Turbomachinery Symposium*, 2009, 139-175.
- [4] Marty, J., Castillon, L., Boniface, J.C., Burguburu, S., Godard A., Numerical and Experimental Investigations of Flow Control in Axial Compressors, *Journal for Aerospace lab*, Issue 6. AL06-09, 2013.
- [5] Hubrich, K., Bölcs, A., Ott, P., Boundary Layer Suction via a Slot in a Transonic Compressor - Numerical Parameter Study and First Experiments. *Proceedings of 2004 ASME Turbo EXPO*, 2004, GT2004-53758.
- [6] Dang, T. Q., Van, Rooij M., Larosiliere, L. M., Design of Aspirated Compressor Blades Using Three-Dimensional Inverse Method, *Proceedings of ASME and International Gas Turbine Institute Turbo Expo*, 2003, GT-2003-38492.
- [7] George, S. D., Aerodynamic Shape Design and Optimization: Status and Trends, *Journal of Aircraft*, Vol. 29, NO. 6, 1992, 1020 – 1026.
- [8] Mengistu, T., Ghaly, W., Global Optimization Methods for the Aerodynamic Shape Design of Transonic Cascades, 2003.
- [9] Girish, M., William, A. C., Dongbin, X., Design Optimization of a High Pressure Turbine Blade using Generalized Polynomial Chaos (gPC), *10th World Congress on Structural and Multi-disciplinary Optimization*, Orlando, Florida – USA, 2013.
- [10] Benini, E., Three Dimensional Multi-Objective Optimization of a Transonic Compressor Rotor, *Journal of Propulsion and Power*, Vol. 20, NO. 3, 2004, 559 – 565, ISSN: 0748 – 4658.
- [11] Ramsden, K.W., *Turbomachinery Design Manuel – Volume 1 and 2* Cranfield University, (Unpublished Thermal Power MSc Course notes), 2008.
- [12] Alstom Power, GT-26 Industrial Engine Specifications [online] Available: <http://www.power.alstom.com>, (March 18th, 2014).
- [13] Rolls-Royce, RB211 GT61 and SK30 GT Industrial Gas Turbine Engine Specification [Online] Available: <http://www.rollsroyce.com>, (April 3rd, 2014).
- [14] LYULKA AL-21F-3, Military Turbojet/Turbofan Specification [Online] Available: <http://www.jet-engine.net/miltspec.html>, (April 12th, 2014).
- [15] Smith, S.F., A Simple Correlation of Turbine Efficiency, *Journal of the Royal Aeronautical Society*, 1969.
- [16] Walsh, P. P., & Fletcher, P., *Gas Turbine Performance*. (2nd ed). Oxford: Blackwell Science, 159-206, 200

Notation and Units

A = the Cross sectional Area of the Annulus (m^2)
 h = the height of the Annulus (m)
 W = mass flow (Kg/s)
 T_3 = turbine entry temperature (K)
 P_3 = Inlet pressure to the turbine (KPa)
 Q = mass flow function (JKg/K)
 PR = Pressure Ratio

Pamt = Ambient Pressure (KPa)
 Dm = Mean diameter of the Annulus (m)
 U = Blade Speed (m/s)
 V = velocity (m/s)
 W = Mass Flow (kg/s)
 BHM = Blade Mid Height

ANNEX - A
Efficiency Correlation Chart
(SINGLE STAGE TURBINES)



Isentropic Flow of Dry

GAS CONSTANT = 287

GAMMA = 1.29

Mach No	T/t	P/p	V/Root.T	1000 Q	1000 q	A/A*
0.000	1.0000	1.0000	0.000	0.000	0.000	Infinity
0.005	1.0000	1.0000	0.006	0.335	0.335	117.1807
0.010	1.0000	1.0001	0.192	0.670	0.870	58.8526
0.015	1.0000	1.0001	0.289	1.005	1.005	39.0547
0.020	1.0001	1.0003	0.385	1.341	1.341	28.3015
0.025	1.0001	1.0004	0.481	1.675	1.675	22.4442
0.030	1.0001	1.0006	0.577	2.010	2.011	18.5399
0.035	1.0002	1.0008	0.673	2.345	2.347	16.7516
0.040	1.0002	1.0010	0.770	2.679	2.682	14.6628
0.045	1.0003	1.0013	0.866	3.013	3.017	13.0350
0.050	1.0004	1.0016	0.962	3.347	3.353	11.7247
0.055	1.0004	1.0020	1.058	3.681	3.688	10.6751
0.060	1.0005	1.0023	1.154	4.014	4.024	9.7851
0.065	1.0005	1.0027	1.250	4.347	4.359	9.0395
0.070	1.0007	1.0032	1.346	4.680	4.695	8.3934
0.075	1.0008	1.0036	1.443	5.012	5.030	7.8371
0.080	1.0009	1.0041	1.539	5.344	5.366	7.3506
0.085	1.0010	1.0047	1.635	5.675	5.702	6.9214
0.090	1.0012	1.0052	1.731	6.006	6.037	6.5402
0.095	1.0013	1.0058	1.827	6.336	6.373	6.1999
0.100	1.0015	1.0065	1.923	6.666	6.709	5.8926
0.105	1.0016	1.0071	2.019	6.995	7.045	5.6153
0.110	1.0018	1.0078	2.115	7.324	7.381	5.3633
0.115	1.0019	1.0086	2.211	7.652	7.717	5.1334
0.120	1.0021	1.0093	2.307	7.979	8.054	4.9228
0.125	1.0023	1.0101	2.402	8.306	8.390	4.7282
0.130	1.0025	1.0109	2.498	8.632	8.726	4.5506
0.135	1.0029	1.0116	2.594	8.957	9.063	4.3854
0.140	1.0028	1.0127	2.690	9.281	9.399	4.2321
0.145	1.0030	1.0135	2.785	9.605	9.735	4.0895
0.150	1.0033	1.0145	2.882	9.928	10.073	3.9585
0.155	1.0035	1.0155	2.977	10.250	10.410	3.8322
0.160	1.0037	1.0165	3.073	10.571	10.747	3.7155
0.165	1.0039	1.0177	3.169	10.891	11.084	3.6065
0.170	1.0042	1.0188	3.264	11.211	11.421	3.5039
0.175	1.0044	1.0199	3.360	11.529	11.759	3.4071
0.180	1.0047	1.0211	3.455	11.848	12.096	3.3158
0.185	1.0050	1.0223	3.551	12.163	12.434	3.2295
0.190	1.0052	1.0235	3.646	12.479	12.771	3.1479
0.195	1.0055	1.0248	3.742	12.793	13.109	3.0705
0.200	1.0058	1.0261	3.837	13.106	13.447	2.9971
0.205	1.0061	1.0274	3.933	13.418	13.785	2.9274
0.210	1.0064	1.0288	4.028	13.729	14.124	2.8611
0.215	1.0067	1.0302	4.123	14.039	14.462	2.7979
0.220	1.0070	1.0316	4.218	14.348	14.801	2.7377
0.225	1.0073	1.0331	4.314	14.655	15.140	2.6803
0.230	1.0077	1.0346	4.409	14.962	15.479	2.6254
0.235	1.0080	1.0361	4.504	15.267	15.818	2.5729
0.240	1.0084	1.0377	4.599	15.571	16.157	2.5227
0.245	1.0087	1.0393	4.694	15.873	16.497	2.4748
0.250	1.0091	1.0409	4.789	16.174	16.837	2.4288

Mach No	T/t	P/p	V/Root.T	1000 Q	1000 q	A/A*
0.255	1.0094	1.0426	4.884	16.472	17.176	2.3844
0.260	1.0098	1.0443	4.978	16.775	17.516	2.3415
0.265	1.0102	1.0461	5.073	17.079	17.857	2.3012
0.270	1.0106	1.0479	5.168	17.382	18.197	2.2620
0.275	1.0110	1.0497	5.263	17.685	18.538	2.2243
0.280	1.0114	1.0516	5.357	17.987	18.879	2.1880
0.285	1.0118	1.0535	5.452	18.289	19.219	2.1530
0.290	1.0122	1.0554	5.546	18.591	19.561	2.1194
0.295	1.0126	1.0574	5.641	18.892	19.902	2.0889
0.300	1.0131	1.0594	5.735	19.193	20.244	2.0598
0.305	1.0135	1.0614	5.829	19.495	20.586	2.0323
0.310	1.0139	1.0635	5.924	19.797	20.928	1.9961
0.315	1.0144	1.0656	6.018	19.998	21.270	1.9610
0.320	1.0148	1.0678	6.112	20.241	21.612	1.9268
0.325	1.0153	1.0699	6.206	20.523	21.955	1.8943
0.330	1.0158	1.0722	6.300	20.767	22.298	1.8627
0.335	1.0163	1.0744	6.394	21.073	22.641	1.8320
0.340	1.0168	1.0767	6.488	21.347	22.985	1.8021
0.345	1.0173	1.0791	6.582	21.619	23.328	1.7729
0.350	1.0178	1.0815	6.675	21.889	23.671	1.7444
0.355	1.0183	1.0839	6.769	22.155	24.017	1.7165
0.360	1.0188	1.0863	6.863	22.423	24.361	1.6891
0.365	1.0193	1.0888	6.956	22.690	24.706	1.6623
0.370	1.0199	1.0914	7.050	22.954	25.051	1.6361
0.375	1.0204	1.0939	7.143	23.215	25.396	1.6104
0.380	1.0209	1.0966	7.236	23.475	25.742	1.5851
0.385	1.0215	1.0992	7.329	23.733	26.087	1.5601
0.390	1.0221	1.1019	7.422	23.989	26.434	1.5354
0.395	1.0226	1.1046	7.515	24.243	26.780	1.5111
0.400	1.0232	1.1074	7.608	24.496	27.127	1.4871
0.405	1.0238	1.1102	7.702	24.746	27.473	1.4634
0.410	1.0244	1.1131	7.795	24.994	27.819	1.4400
0.415	1.0250	1.1160	7.887	25.241	28.166	1.4168
0.420	1.0256	1.1189	7.980	25.486	28.511	1.3938
0.425	1.0262	1.1219	8.073	25.729	28.856	1.3710
0.430	1.0268	1.1249	8.165	25.969	29.202	1.3484
0.435	1.0274	1.1280	8.257	26.208	29.547	1.3260
0.440	1.0281	1.1311	8.350	26.444	29.892	1.3038
0.445	1.0287	1.1342	8.442	26.679	30.237	1.2818
0.450	1.0294	1.1374	8.534	26.912	30.582	1.2599
0.455	1.0300	1.1406	8.626	27.142	30.927	1.2381
0.460	1.0307	1.1438	8.718	27.371	31.271	1.2164
0.465	1.0314	1.1472	8.810	27.598	31.615	1.1948
0.470	1.0320	1.1506	8.902	27.822	31.958	1.1733
0.475	1.0327	1.1540	8.994	28.044	32.302	1.1518
0.480	1.0334	1.1574	9.085	28.265	32.714	1.1303
0.485	1.0341	1.1609	9.177	28.483	33.066	1.1088
0.490	1.0348	1.1644	9.268	28.698	33.418	1.0873
0.495	1.0356	1.1680	9.360	28.913	33.771	1.0658
0.500	1.0363	1.1716	9.451	29.125	34.124	1.0443

GAS CONSTANT = 287

Mach No	T/t	P/p	V/Root.T	1000 Q	1000 q	A/A*
0.600	1.3353	9.457	29.125	34.124	1.3457	
0.505	1.5370	1.1753	9.542	29.335	34.477	1.3360
0.510	1.5377	1.1760	9.553	29.542	34.631	1.3266
0.515	1.5385	1.1769	9.564	29.748	35.185	1.3174
0.520	1.5392	1.1778	9.575	29.951	35.539	1.3115
0.525	1.5400	1.1784	9.586	30.152	35.884	1.3072
0.530	1.5407	1.1793	9.598	30.351	36.249	1.2942
0.535	1.5415	1.1803	9.609	30.548	36.625	1.2859
0.540	1.5423	1.2027	10.177	30.742	36.961	1.2777
0.545	1.5431	1.2053	10.268	30.935	37.317	1.2698
0.550	1.5439	1.2104	10.358	31.125	37.674	1.2620
0.555	1.5447	1.2145	10.448	31.313	38.031	1.2544
0.560	1.5455	1.2187	10.538	31.499	38.388	1.2470
0.565	1.5463	1.2230	10.628	31.682	38.745	1.2398
0.570	1.5471	1.2272	10.718	31.863	39.102	1.2326
0.575	1.5479	1.2315	10.808	32.043	39.463	1.2259
0.580	1.5488	1.2360	10.897	32.219	39.822	1.2191
0.585	1.5496	1.2404	10.987	32.394	40.182	1.2126
0.590	1.5505	1.2449	11.076	32.565	40.541	1.2062
0.595	1.5513	1.2494	11.165	32.737	40.902	1.1999
0.600	1.5522	1.2540	11.255	32.904	41.262	1.1938
0.605	1.5531	1.2588	11.344	33.070	41.624	1.1878
0.610	1.5540	1.2633	11.433	33.234	41.985	1.1818
0.615	1.5548	1.2681	11.522	33.395	42.347	1.1757
0.620	1.5557	1.2729	11.610	33.554	42.709	1.1697
0.625	1.5566	1.2777	11.699	33.710	43.072	1.1637
0.630	1.5575	1.2825	11.787	33.865	43.436	1.1577
0.635	1.5584	1.2873	11.875	34.017	43.799	1.1517
0.640	1.5594	1.2922	11.964	34.167	44.163	1.1457
0.645	1.5603	1.2970	12.051	34.314	44.528	1.1397
0.650	1.5613	1.3019	12.141	34.462	44.893	1.1337
0.655	1.5622	1.3079	12.228	34.603	45.258	1.1277
0.660	1.5632	1.3128	12.318	34.744	45.624	1.1217
0.665	1.5641	1.3178	12.404	34.882	45.991	1.1157
0.670	1.5651	1.3228	12.492	35.021	46.357	1.1097
0.675	1.5661	1.3278	12.579	35.153	46.722	1.1037
0.680	1.5670	1.3328	12.668	35.285	47.087	1.1032
0.685	1.5680	1.3378	12.754	35.416	47.451	1.0972
0.690	1.5690	1.3427	12.841	35.541	47.815	1.0912
0.695	1.5700	1.3476	12.928	35.667	48.179	1.0852
0.700	1.5711	1.3525	13.015	35.792	48.543	1.0792
0.705	1.5721	1.3574	13.101	35.917	48.907	1.0732
0.710	1.5731	1.3623	13.188	36.042	49.271	1.0672
0.715	1.5741	1.3672	13.274	36.167	49.635	1.0612
0.720	1.5752	1.3721	13.361	36.292	50.000	1.0552
0.725	1.5762	1.3770	13.447	36.417	50.364	1.0492
0.730	1.5773	1.3819	13.533	36.542	50.728	1.0432
0.735	1.5783	1.3868	13.619	36.667	51.100	1.0372
0.740	1.5794	1.4048	13.705	36.792	51.464	1.0312
0.745	1.5805	1.4116	13.791	36.917	51.828	1.0252
0.750	1.5816	1.4175	13.878	37.042	52.192	1.0192

GAMMA = 1.29

Mach No	T/t	P/p	V/Root.T	1000 Q	1000 q	A/A*
0.755	1.5827	1.4237	13.962	37.167	52.556	1.0132
0.760	1.5838	1.4291	14.047	37.292	52.919	1.0072
0.765	1.5849	1.4346	14.132	37.417	53.282	1.0012
0.770	1.5860	1.4402	14.217	37.542	53.645	0.9952
0.775	1.5871	1.4458	14.302	37.667	54.008	0.9892
0.780	1.5882	1.4515	14.387	37.792	54.371	0.9832
0.785	1.5894	1.4573	14.472	37.917	54.734	0.9772
0.790	1.5905	1.4631	14.557	38.042	55.097	0.9712
0.795	1.5916	1.4690	14.642	38.167	55.460	0.9652
0.800	1.5928	1.4748	14.727	38.292	55.823	0.9592
0.805	1.5940	1.4807	14.812	38.417	56.186	0.9532
0.810	1.5951	1.4866	14.897	38.542	56.549	0.9472
0.815	1.5963	1.4925	14.982	38.667	56.912	0.9412
0.820	1.5975	1.4984	15.067	38.792	57.275	0.9352
0.825	1.5987	1.5043	15.152	38.917	57.638	0.9292
0.830	1.5999	1.5102	15.237	39.042	58.001	0.9232
0.835	1.6011	1.5161	15.322	39.167	58.364	0.9172
0.840	1.6023	1.5220	15.407	39.292	58.727	0.9112
0.845	1.6035	1.5279	15.492	39.417	59.090	0.9052
0.850	1.6047	1.5338	15.577	39.542	59.453	0.8992
0.855	1.6059	1.5397	15.662	39.667	59.816	0.8932
0.860	1.6071	1.5456	15.747	39.792	60.179	0.8872
0.865	1.6083	1.5515	15.832	39.917	60.542	0.8812
0.870	1.6095	1.5574	15.917	40.042	60.905	0.8752
0.875	1.6107	1.5633	16.002	40.167	61.268	0.8692
0.880	1.6119	1.5692	16.087	40.292	61.631	0.8632
0.885	1.6131	1.5751	16.172	40.417	61.994	0.8572
0.890	1.6143	1.5810	16.257	40.542	62.357	0.8512
0.895	1.6155	1.5869	16.342	40.667	62.720	0.8452
0.900	1.6167	1.5928	16.427	40.792	63.083	0.8392
0.905	1.6179	1.5987	16.512	40.917	63.446	0.8332
0.910	1.6191	1.6046	16.597	41.042	63.809	0.8272
0.915	1.6203	1.6105	16.682	41.167	64.172	0.8212
0.920	1.6215	1.6164	16.767	41.292	64.535	0.8152
0.925	1.6227	1.6223	16.852	41.417	64.898	0.8092
0.930	1.6239	1.6282	16.937	41.542	65.261	0.8032
0.935	1.6251	1.6341	17.022	41.667	65.624	0.7972
0.940	1.6263	1.6400	17.107	41.792	65.987	0.7912
0.945	1.6275	1.6459	17.192	41.917	66.350	0.7852
0.950	1.6287	1.6518	17.277	42.042	66.713	0.7792
0.955	1.6299	1.6577	17.362	42.167	67.076	0.7732
0.960	1.6311	1.6636	17.447	42.292	67.439	0.7672
0.965	1.6323	1.6695	17.532	42.417	67.802	0.7612
0.970	1.6335	1.6754	17.617	42.542	68.165	0.7552
0.975	1.6347	1.6813	17.702	42.667	68.528	0.7492
0.980	1.6359	1.6872	17.787	42.792	68.891	0.7432
0.985	1.6371	1.6931	17.872	42.917	69.254	0.7372
0.990	1.6383	1.6990	17.957	43.042	69.617	0.7312
0.995	1.6395	1.7049	18.042	43.167	70.000	0.7252
1.000	1.6407	1.7108	18.127	43.292	70.363	0.7192

GAS CONSTANT = 287

Mach No	T/t	P/p	V/Root.T	1000 Q	1000 q	A/A*
0.600	1.0000	1.0000	0.000	0.000	0.000	Infinity
0.605	1.0000	1.0000	0.007	0.339	0.339	116.7820
0.610	1.0000	1.0001	0.195	0.878	0.878	58.3936
0.615	1.0000	1.0001	0.292	1.017	1.017	38.9319
0.620	1.0001	1.0003	0.389	1.356	1.356	29.2015
0.625	1.0001	1.0004	0.487	1.695	1.695	23.3645
0.630	1.0001	1.0006	0.584	2.033	2.033	19.4736
0.635	1.0002	1.0008	0.681	2.372	2.372	16.6846
0.640	1.0003	1.0011	0.778	2.710	2.710	14.6111
0.645	1.0003	1.0013	0.876	3.048	3.048	12.9908
0.650	1.0004	1.0017	0.973	3.386	3.386	11.6950
0.655	1.0005	1.0020	1.070	3.723	3.723	10.6850
0.660	1.0006	1.0024	1.167	4.061	4.061	9.7520
0.665	1.0007	1.0028	1.265	4.397	4.397	8.9051
0.670	1.0008	1.0032	1.362	4.734	4.734	8.1352
0.675	1.0009	1.0037	1.459	5.070	5.070	7.4308
0.680	1.0010	1.0042	1.556	5.405	5.405	6.7859
0.685	1.0011	1.0046	1.653	5.740	5.740	6.1963
0.690	1.0013	1.0054	1.751	6.075	6.108	5.6583
0.695	1.0014	1.0060	1.848	6.409	6.447	5.1676
0.700	1.0016	1.0068	1.945	6.743	6.787	4.7200
0.705	1.0018	1.0073	2.042	7.076	7.127	4.3126
0.710	1.0019	1.0080	2.139	7.408	7.467	3.9415
0.715	1.0021	1.0088	2.236	7.740	7.807	3.6025
0.720	1.0023	1.0096	2.333	8.071	8.148	3.2920
0.725	1.0025	1.0104	2.430	8.401	8.486	3.0062
0.730	1.0027	1.0112	2.527	8.730	8.826	2.7407
0.735	1.0029	1.0121	2.624	9.059	9.169	2.4911
0.740	1.0031	1.0130	2.721	9.387	9.509	2.2533
0.745	1.0034	1.0139	2.818	9.715	9.850	2.0331
0.750	1.0036	1.0148	2.914	10.041	10.191	1.8357
0.755	1.0038	1.0156	3.011	10.367	10.532	1.6561
0.760	1.0041	1.0170	3.108	10.691	10.873	1.4900
0.765	1.0044	1.0181	3.205	11.015	11.214	1.3350
0.770	1.0046	1.0192	3.303	11.339	11.556	1.1900
0.775	1.0049	1.0204	3.398	11.660	11.897	1.0530
0.780	1.0052	1.0216	3.494	11.981	12.239	0.9230
0.785	1.0055	1.0228	3.591	12.300	12.581	0.8000
0.790	1.0058	1.0240	3.687	12.619	12.923	0.6830
0.795	1.0061	1.0253	3.784	12.937	13.265	0.5720
0.800	1.0064	1.0267	3.880	13.254	13.607	0.4660
0.805	1.0067	1.0280	3.977	13.569	13.949	0.3650
0.810	1.0071	1.0294	4.073	13.883	14.292	0.2690
0.815	1.0074	1.0309	4.169	14.197	14.635	0.1780
0.820	1.0077	1.0323	4.265	14.509	14.978	0.0920
0.825	1.0081	1.0338	4.362	14.819	15.321	0.0110
0.830						

GAS CONSTANT = 287

GAMMA = 1.32

Mach No	T/T	P/P	V/RootT	1000 Q	1000 q	A/A*
0.500	1.0400	1.1795	9.543	29.415	34.581	1.3462
0.505	1.0408	1.1794	9.635	29.626	34.940	1.3386
0.510	1.0416	1.1792	9.728	29.839	35.300	1.3313
0.515	1.0424	1.1790	9.818	30.046	35.660	1.3241
0.520	1.0433	1.1789	9.909	30.248	36.020	1.3170
0.525	1.0441	1.1788	10.000	30.448	36.381	1.3100
0.530	1.0449	1.1786	10.092	30.646	36.742	1.3030
0.535	1.0458	1.1785	10.183	30.845	37.104	1.2960
0.540	1.0467	1.1784	10.274	31.042	37.468	1.2890
0.545	1.0475	1.1782	10.364	31.238	37.829	1.2820
0.550	1.0484	1.1781	10.455	31.427	38.192	1.2750
0.555	1.0493	1.1780	10.546	31.618	38.555	1.2680
0.560	1.0502	1.1779	10.636	31.802	38.919	1.2610
0.565	1.0511	1.1778	10.727	31.987	39.284	1.2540
0.570	1.0520	1.1777	10.817	32.169	39.648	1.2470
0.575	1.0529	1.1776	10.907	32.349	40.014	1.2400
0.580	1.0538	1.1775	10.997	32.527	40.379	1.2330
0.585	1.0548	1.1774	11.087	32.702	40.745	1.2260
0.590	1.0557	1.1773	11.177	32.875	41.112	1.2190
0.595	1.0566	1.1772	11.268	33.046	41.479	1.2120
0.600	1.0576	1.1771	11.358	33.216	41.846	1.2050
0.605	1.0585	1.1770	11.448	33.381	42.214	1.1980
0.610	1.0595	1.1769	11.538	33.546	42.583	1.1910
0.615	1.0604	1.1768	11.628	33.707	42.952	1.1840
0.620	1.0614	1.1767	11.718	33.867	43.321	1.1770
0.625	1.0623	1.1766	11.808	34.024	43.691	1.1700
0.630	1.0633	1.1765	11.899	34.179	44.051	1.1630
0.635	1.0643	1.1764	11.979	34.332	44.432	1.1560
0.640	1.0653	1.1763	12.068	34.482	44.803	1.1490
0.645	1.0663	1.1762	12.158	34.630	45.175	1.1420
0.650	1.0673	1.1761	12.248	34.778	45.547	1.1350
0.655	1.0683	1.1760	12.338	34.919	45.920	1.1280
0.660	1.0693	1.1759	12.428	35.051	46.294	1.1210
0.665	1.0703	1.1758	12.518	35.190	46.667	1.1140
0.670	1.0713	1.1757	12.608	35.328	47.042	1.1070
0.675	1.0723	1.1756	12.698	35.467	47.418	1.1000
0.680	1.0733	1.1755	12.788	35.605	47.792	1.0930
0.685	1.0743	1.1754	12.878	35.743	48.168	1.0860
0.690	1.0753	1.1753	12.968	35.881	48.544	1.0790
0.695	1.0763	1.1752	13.058	36.018	48.921	1.0720
0.700	1.0773	1.1751	13.148	36.155	49.297	1.0650
0.705	1.0783	1.1750	13.238	36.292	49.672	1.0580
0.710	1.0793	1.1749	13.328	36.429	50.048	1.0510
0.715	1.0803	1.1748	13.418	36.565	50.424	1.0440
0.720	1.0813	1.1747	13.508	36.702	50.800	1.0370
0.725	1.0823	1.1746	13.598	36.838	51.176	1.0300
0.730	1.0833	1.1745	13.688	36.974	51.552	1.0230
0.735	1.0843	1.1744	13.778	37.110	51.928	1.0160
0.740	1.0853	1.1743	13.868	37.246	52.304	1.0090
0.745	1.0863	1.1742	13.958	37.382	52.680	1.0020
0.750	1.0873	1.1741	14.048	37.518	53.056	0.9950

Mach No	T/T	P/P	V/RootT	1000 Q	1000 q	A/A*
0.755	1.0883	1.1740	14.138	37.654	53.432	0.9880
0.760	1.0893	1.1739	14.228	37.790	53.808	0.9810
0.765	1.0903	1.1738	14.318	37.926	54.184	0.9740
0.770	1.0913	1.1737	14.408	38.062	54.560	0.9670
0.775	1.0923	1.1736	14.498	38.198	54.936	0.9600
0.780	1.0933	1.1735	14.588	38.334	55.312	0.9530
0.785	1.0943	1.1734	14.678	38.470	55.688	0.9460
0.790	1.0953	1.1733	14.768	38.606	56.064	0.9390
0.795	1.0963	1.1732	14.858	38.742	56.440	0.9320
0.800	1.0973	1.1731	14.948	38.878	56.816	0.9250
0.805	1.0983	1.1730	15.038	39.014	57.192	0.9180
0.810	1.0993	1.1729	15.128	39.150	57.568	0.9110
0.815	1.1003	1.1728	15.218	39.286	57.944	0.9040
0.820	1.1013	1.1727	15.308	39.422	58.320	0.8970
0.825	1.1023	1.1726	15.398	39.558	58.696	0.8900
0.830	1.1033	1.1725	15.488	39.694	59.072	0.8830
0.835	1.1043	1.1724	15.578	39.830	59.448	0.8760
0.840	1.1053	1.1723	15.668	39.966	59.824	0.8690
0.845	1.1063	1.1722	15.758	40.102	60.200	0.8620
0.850	1.1073	1.1721	15.848	40.238	60.576	0.8550
0.855	1.1083	1.1720	15.938	40.374	60.952	0.8480
0.860	1.1093	1.1719	16.028	40.510	61.328	0.8410
0.865	1.1103	1.1718	16.118	40.646	61.704	0.8340
0.870	1.1113	1.1717	16.208	40.782	62.080	0.8270
0.875	1.1123	1.1716	16.298	40.918	62.456	0.8200
0.880	1.1133	1.1715	16.388	41.054	62.832	0.8130
0.885	1.1143	1.1714	16.478	41.190	63.208	0.8060
0.890	1.1153	1.1713	16.568	41.326	63.584	0.7990
0.895	1.1163	1.1712	16.658	41.462	63.960	0.7920
0.900	1.1173	1.1711	16.748	41.598	64.336	0.7850
0.905	1.1183	1.1710	16.838	41.734	64.712	0.7780
0.910	1.1193	1.1709	16.928	41.870	65.088	0.7710
0.915	1.1203	1.1708	17.018	42.006	65.464	0.7640
0.920	1.1213	1.1707	17.108	42.142	65.840	0.7570
0.925	1.1223	1.1706	17.198	42.278	66.216	0.7500
0.930	1.1233	1.1705	17.288	42.414	66.592	0.7430
0.935	1.1243	1.1704	17.378	42.550	66.968	0.7360
0.940	1.1253	1.1703	17.468	42.686	67.344	0.7290
0.945	1.1263	1.1702	17.558	42.822	67.720	0.7220
0.950	1.1273	1.1701	17.648	42.958	68.096	0.7150
0.955	1.1283	1.1700	17.738	43.094	68.472	0.7080
0.960	1.1293	1.1699	17.828	43.230	68.848	0.7010
0.965	1.1303	1.1698	17.918	43.366	69.224	0.6940
0.970	1.1313	1.1697	18.008	43.502	69.600	0.6870
0.975	1.1323	1.1696	18.098	43.638	69.976	0.6800
0.980	1.1333	1.1695	18.188	43.774	70.352	0.6730
0.985	1.1343	1.1694	18.278	43.910	70.728	0.6660
0.990	1.1353	1.1693	18.368	44.046	71.104	0.6590
0.995	1.1363	1.1692	18.458	44.182	71.480	0.6520
1.000	1.1373	1.1691	18.548	44.318	71.856	0.6450

GAS CONSTANT = 287

GAMMA = 1.4

Mach No	T/T	P/P	V/RootT	1000 Q	1000 q	A/A*
0.500	1.0000	1.0000	0.693	0.500	0.630	infinity
0.505	1.0000	1.0000	0.700	0.509	0.649	115.7425
0.510	1.0000	1.0001	0.707	0.518	0.668	57.8738
0.515	1.0000	1.0002	0.714	0.527	0.687	38.5856
0.520	1.0001	1.0003	0.721	0.536	0.706	28.9421
0.525	1.0001	1.0004	0.728	0.545	0.725	23.1568
0.530	1.0002	1.0005	0.735	0.554	0.744	19.3005
0.535	1.0002	1.0006	0.742	0.563	0.763	16.3466
0.540	1.0003	1.0007	0.749	0.572	0.782	13.8777
0.545	1.0003	1.0008	0.756	0.581	0.801	11.5914
0.550	1.0004	1.0009	0.763	0.590	0.820	9.5858
0.555	1.0004	1.0010	0.770	0.599	0.839	7.8601
0.560	1.0005	1.0011	0.777	0.608	0.858	6.3843
0.565	1.0005	1.0012	0.784	0.617	0.877	5.1984
0.570	1.0006	1.0013	0.791	0.626	0.896	4.2526
0.575	1.0006	1.0014	0.798	0.635	0.915	3.4979
0.580	1.0007	1.0015	0.805	0.644	0.934	2.8843
0.585	1.0007	1.0016	0.812	0.653	0.953	2.3628
0.590	1.0008	1.0017	0.819	0.662	0.972	1.9043
0.595	1.0008	1.0018	0.826	0.671	0.991	1.4898
0.600	1.0009	1.0019	0.833	0.680	1.010	1.1103
0.605	1.0009	1.0020	0.840	0.689	1.029	0.7688
0.610	1.0010	1.0021	0.847	0.698	1.048	0.5583
0.615	1.0010	1.0022	0.854	0.707	1.067	0.4119
0.620	1.0011	1.0023	0.861	0.716	1.086	0.2944
0.625	1.0011	1.0024	0.868	0.725	1.105	0.2000
0.630	1.0012	1.0025	0.875	0.734	1.124	0.1325
0.635	1.0012	1.0026	0.882	0.743	1.143	0.0867
0.640	1.0013	1.0027	0.889	0.752	1.162	0.0564
0.645	1.0013	1.0028	0.896	0.761	1.181	0.0361
0.650	1.0014	1.0029	0.903	0.770	1.200	0.0217
0.655	1.0014	1.0030	0.910	0.779	1.219	0.0125
0.660	1.0015	1.0031	0.917	0.788	1.238	0.0066
0.665	1.0015	1.0032	0.924	0.797	1.257	0.0033
0.670	1.0016	1.0033	0.931	0.806	1.276	0.0016
0.675	1.0016	1.0034	0.938	0.815	1.295	0.0007
0.680	1.0017	1.0035	0.945	0.824	1.314	0.0003
0.685	1.0017	1.0036	0.952	0.833	1.333	0.0001
0.690	1.0018	1.0037	0.959	0.842	1.352	0.0000
0.695	1.0018	1.0038	0.966	0.851	1.371	0.0000
0.700	1.0019	1.0039	0.973	0.860	1.390	0.0000
0.705	1.0019	1.0040	0.980	0.869	1.409	0.0000
0.710	1.0020	1.0041	0.987	0.878	1.428	0.0000
0.715	1.0020	1.0042	0.994	0.887	1.447	0.0000
0.720	1.0021	1.0043	1.001	0.896	1.466	0.0000
0.725	1.0021	1.0044	1.008	0.905	1.485	0.0000